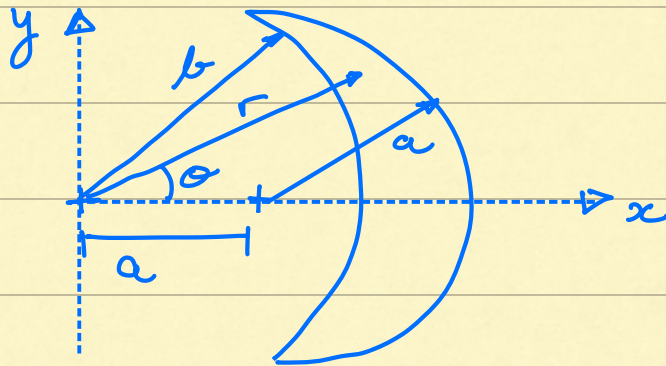


3C7/2026

1.



(a) Inner surface $r = b$

$$\psi = C \left(1 - \frac{b^2}{b^2}\right) (b^2 - 2ab \cos \theta) = 0 \quad \checkmark$$

Outer surface $r = 2a \cos \theta$

$$\psi = \left(1 - \frac{b^2}{4a^2 \cos^2 \theta}\right) (4a^2 \cos^2 \theta - 4a^2 \cos^2 \theta) = 0 \quad \checkmark$$

$$\nabla^2 \psi = \left(\frac{\partial^2 \psi}{\partial r^2} + \frac{1}{r} \frac{\partial \psi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \psi}{\partial \theta^2} \right)$$

$$\frac{\partial \psi}{\partial r} = C \left(2r - 2a \cos \theta - \frac{2ab^2 \cos \theta}{r^2} \right)$$

$$\frac{\partial^2 \psi}{\partial r^2} = C \left(2 + \frac{4ab^2 \cos \theta}{r^3} \right)$$

$$\frac{\partial^2 \psi}{\partial \theta^2} = 2C \left(1 - \frac{b^2}{r^2} \right) ar \cos \theta$$

$$\Rightarrow \nabla^2 \psi = C \left[2 + \frac{4ab^2}{r^3} \cos \theta + \right. \\ \left. 2 - \frac{2a}{r} \cos \theta - \frac{2ab^2}{r^3} \cos \theta + \right. \\ \left. \frac{2a}{r} \cos \theta - \frac{2ab^2}{r^3} \cos \theta \right]$$

$$= 4C = -2G\alpha$$

$$\text{ie } C = -\frac{G\alpha}{2}$$

$\psi = 0$ on boundary & $\nabla^2 \psi = \text{constant} \Rightarrow$
 $\psi = \text{Prandtl stress function.}$

(b) In cartesian co-ordinates

$$\psi = -\frac{G\alpha}{2} \left[1 - \frac{b^2}{x^2 + y^2} \right] [x^2 + y^2 - 2ax]$$

$$= -\frac{G\alpha}{2} \left[x^2 + y^2 + \frac{2ab^2x}{x^2 + y^2} - 2ax - b^2 \right]$$

$$\sigma_{zy} = -\frac{\partial \psi}{\partial x}$$

$$\sigma_{zy} = G\alpha \left[x - a + \frac{ab^2}{x^2 + y^2} - \frac{2ab^2x^2}{(x^2 + y^2)^2} \right]$$

$$(c) \quad \sigma_{zy}(b, 0) = G\alpha \left[b - a + a - 2a \right]$$

$$= -G\alpha a \left(2 - \frac{b}{a} \right)$$

$$\sigma_{zy}(2a, 0) = G\alpha \left[2a - a + \frac{b^2}{4a} - \frac{b^2}{2a} \right]$$

$$= G\alpha a \left[1 - \frac{b^2}{4a^2} \right]$$

$$(d) \quad \text{let } \frac{b}{a} = 2 - \beta$$

$$\Rightarrow \sigma_{zy}(b, 0) = -G\alpha a \beta$$

$$\sigma_{zy}(2a, 0) = G\alpha a \left(1 - \frac{b}{2a} \right) \left(1 + \frac{b}{2a} \right)$$

$$= G\alpha a \left(1 - \frac{2-\beta}{2} \right) \left(1 + \frac{2-\beta}{2} \right)$$

$$= G\alpha a \left(\beta - \frac{\beta^2}{4} \right)$$

$$\Rightarrow |\sigma_{zy}(b, 0)| > |\sigma_{zy}(2a, 0)|$$

2.

(a) Thin disk $\Rightarrow$ plane stress.

For annulus Lamé equations

$$\sigma_r = A - \frac{B}{r^2}, \quad \sigma_\theta = A + \frac{B}{r^2}$$

$$\sigma_r(a) = -p = A - \frac{B}{a^2}$$

$$\sigma_r(b) = 0 = A - \frac{B}{b^2}$$

$$\Rightarrow A = \frac{B}{b^2} \quad \& \quad B = \frac{pa^2b^2}{b^2 - a^2}, \quad A = \frac{pa^2}{b^2 - a^2}$$

$$\sigma_r = \frac{pa^2}{b^2 - a^2} \left(1 - \frac{b^2}{r^2} \right) = \frac{-p}{1 - \frac{b^2}{a^2}} \left(1 - \frac{b^2}{r^2} \right)$$

$$\sigma_\theta = \frac{-p}{1 - \frac{b^2}{a^2}} \left(1 + \frac{b^2}{r^2} \right)$$

$$\epsilon_\theta = \frac{1}{E} (\sigma_\theta - \nu \sigma_r) \quad \because \sigma_z = 0$$

$$\Rightarrow \epsilon_\theta = \frac{u}{r} = -\frac{p}{E(1-\frac{b^2}{a^2})} \left[1 + \frac{b^2}{r^2} - \nu \left(1 - \frac{b^2}{r^2} \right) \right]$$

$$\Rightarrow \frac{u(a)}{a} = \frac{p}{E} \left[\frac{b^2+a^2}{b^2-a^2} + \nu \right]$$

(b) Balance of forces gives a radial pressure $p/2$ on solid cylinder & p on annulus.

Solid cylinder $\sigma_r = \sigma_\theta = -\frac{p}{2}$

$$\epsilon_\theta(a) = \frac{u_c(a)}{a} = -\frac{p}{2E} (1-\nu)$$

Interface compatibility $\Rightarrow$

$$u(a) = u_c(a) + \delta$$

$$\Rightarrow p \left[\frac{b^2+a^2}{b^2-a^2} + \nu \right] = -\frac{p}{2} (1-\nu) + \frac{\delta E}{a}$$

For $b/a = 2$ & $\nu = \frac{1}{3}$ $p = \frac{3}{7} \frac{\delta E}{a}$

(c) Yield initiates at inner radius a of annulus. The active Tresca surface is

$$\sigma_\theta - \sigma_r = - \frac{2p k^2/r^2}{1 - k^2/a^2} = \gamma$$

$$\text{at } r = a \quad p = \frac{3\gamma}{8} = \frac{3SE}{7a}$$

$$\Rightarrow \frac{\delta}{a} = \frac{7}{8} \frac{\gamma}{E}$$

(d) This is an overestimate of the allowable since it neglects the out-of-plane shear stresses due to the difference in thickness between the annulus & solid cylinder.

3.

(a)

$$\phi = B r^n \theta$$

$$\nabla^2 \phi = \left(\frac{\partial^2 \phi}{\partial r^2} + \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} \right)$$

$$= B \left[n(n-1) + n \right] r^{n-2} \theta$$

$$= B n^2 r^{n-2} \theta$$

$$\nabla^4 \phi = B \left[(n-2)(n-3) + (n-2) \right] n^2 r^{n-4} \theta = 0$$

$$\Rightarrow B n^2 (n-2)^2 r^{n-4} \theta = 0$$

$$n = (0, 2).$$

(b) $\sigma_{rr} = \frac{1}{r} \frac{\partial \phi_2}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi_2}{\partial \theta^2} = 2B_2 \theta$

$$\sigma_{\theta\theta} = \frac{\partial^2 \phi_2}{\partial r^2} = 2B_2 \theta$$

$$\sigma_{r\theta} = -\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial \phi_2}{\partial \theta} \right) = -B_2$$

$$\Rightarrow \sigma_{\theta\theta}(r, 0) = 0, \quad \sigma_{r\theta}(r, 0) = -B_2$$

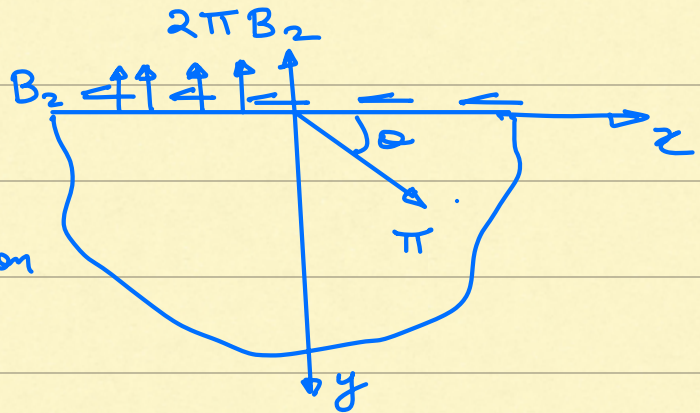
$$\sigma_{\theta\theta}(r, \pi) = 2\pi B_2 \quad \& \quad \sigma_{r\theta}(r, \pi) = -B_2$$

$\Rightarrow$

ie surface $y=0$ loaded with uniform shear traction

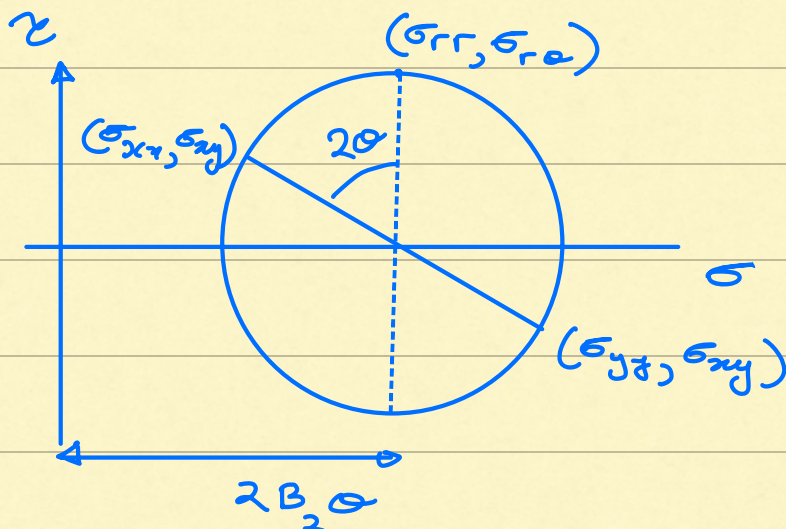
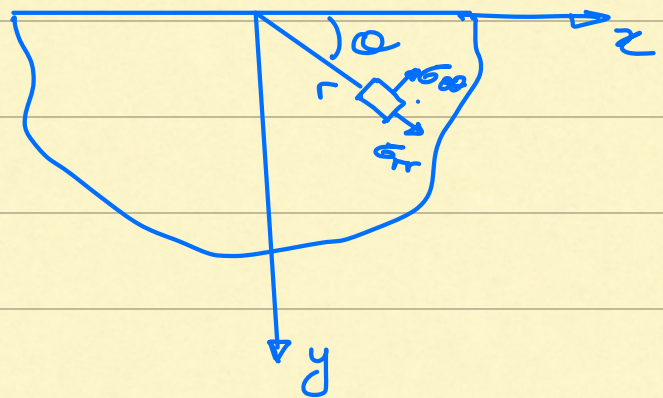
$-B_2$ & normal traction

$2\pi B_2$ on $x < 0$.



(c)

Converting to Cartesian co-ordinates



$$\sigma_{xx} = B_2 (2\theta - \sin 2\theta)$$

$$\sigma_{yy} = B_2 (2\theta + \sin 2\theta)$$

$$\sigma_{xy} = -B_2 \cos 2\theta$$

Recalling $\sin 2\theta = 2 \sin \theta \cos \theta$

ie $\sigma_{xx} = B_2 \left[2 \tan^{-1}\left(\frac{y}{x}\right) - \frac{2xy}{x^2+y^2} \right]$

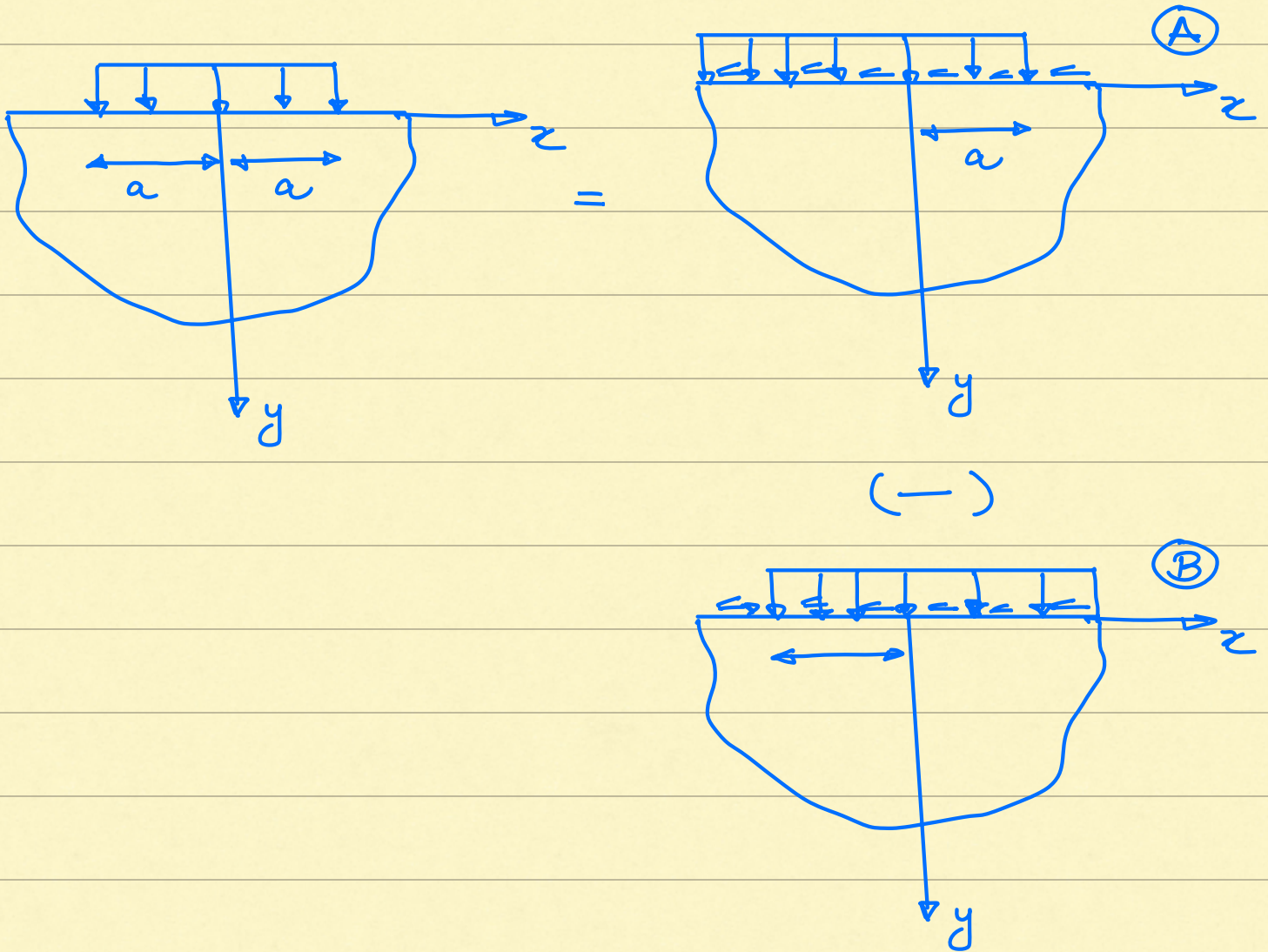
$$\sigma_{yy} = B_2 \left[2 \tan^{-1}\left(\frac{y}{x}\right) + \frac{2xy}{x^2+y^2} \right]$$

& $\cos 2\theta = \cos^2 \theta - \sin^2 \theta$

$$\Rightarrow \sigma_{xy} = -B_2 \left[\frac{x^2 - y^2}{x^2 + y^2} \right]$$

(d)

The problem under consideration now is



Shear stress in (A)

$$\tau_{xy}^{(A)} = -B_2 \left[\frac{(x-a)^2 - y^2}{(x-a)^2 + y^2} \right]$$

Shear stress in (B)

$$\sigma_{xy}^{(B)} = -B_2 \left[\frac{(x+a)^2 - y^2}{(x+a)^2 + y^2} \right]$$

$$\Rightarrow \sigma_{xy} = \sigma_{xy}^{(A)} - \sigma_{xy}^{(B)}$$

$$= -B_2 \left[\frac{(x-a)^2 - y^2}{(x-a)^2 + y^2} - \frac{(x+a)^2 - y^2}{(x+a)^2 + y^2} \right]$$

$$\begin{aligned} \sigma_{yy}(x, 0) &= 2\pi B_2 \quad \text{on } -a \leq x \leq a \\ &= 0 \quad |x| > a \end{aligned}$$

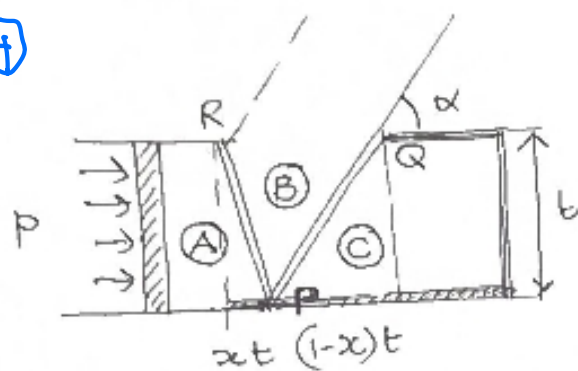
$$\Rightarrow 2\pi B_2 = -p \quad \therefore B_2 = -\frac{p}{2\pi}$$

$$\sigma_{xy}(x, y) = \frac{p}{2\pi} \left[\frac{(x-a)^2 - y^2}{(x-a)^2 + y^2} - \frac{(x+a)^2 - y^2}{(x+a)^2 + y^2} \right]$$

$$\xi \epsilon_{xy}(x, a) = \frac{p}{2\pi} \left[\frac{(x-a)^2 - a^2}{(x-a)^2 + a^2} - \frac{(x+a)^2 - a^2}{(x+a)^2 + a^2} \right]$$

$$= -\frac{p}{a\pi} \frac{4a^3 x}{(x^2 + 2a^2)^2 - 4a^2 x}$$

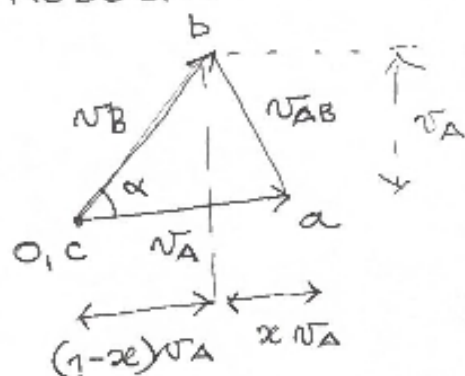
④



$$\overline{PQ} = t \sqrt{(1-x)^2 + 1} = t \sqrt{x^2 - 2x + 2}$$

$$\overline{PR} = t \sqrt{1+x^2}$$

HODOGRAPH



$$N_B = N_A \sqrt{x^2 - 2x + 2}$$

$$N_{AB} = N_A \sqrt{1+x^2}$$

$$W_{ext} = P \cdot t \cdot N_A$$

$$\begin{aligned} W_{int} &= K \left[N_B \cdot \overline{PQ} + N_{AB} \cdot \overline{PR} + f x t \right] = \\ &= K t N_A \left[x^2 - 2x + 2 + 1 + x^2 + f x \right] = \\ &= K t N_A \left[2x^2 - (2-f)x + 3 \right] \end{aligned}$$

$$W_{ext} = W_{int}$$

$$P \cdot t \cdot N_A = K t N_A \left[2x^2 - (2-f)x + 3 \right]$$

$$P = K \left[2x^2 - (2-f)x + 3 \right]$$

$$\frac{\partial P}{\partial x} = 0 \quad K \left[4x - (2-f) \right] = 0 \Rightarrow x = \frac{2-f}{4}$$

(i) if $f = 0$

$$x = \frac{1}{2} \quad (1-x) = \frac{1}{2} \quad \tan \alpha = \frac{2\cancel{f}}{\cancel{f}} = 2$$

hence $\alpha = \tan^{-1}(2)$

(ii) if $f = 0.2$ $x = \frac{1.8}{4} = 0.45$

$$(iii) F = pt = kt \left[\frac{1}{2} \frac{(2-f)^2}{1.68} - \frac{(2-f)^2}{4} + 3 \right] =$$

$$F = kt \left[3 - \frac{(2-f)^2}{8} \right] = 2.68 \, kt$$